



# Comparison of Tone Mode Measurements for a Forward Swept and Baseline Rotor Fan

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# COMPARISON OF TONE MODE MEASUREMENTS FOR A FORWARD SWEEP AND BASELINE ROTOR FAN

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## ABSTRACT

A forward swept fan, designated the Quiet High Speed Fan (QHSF), was tested in the NASA Glenn 9- by 15-foot Low Speed Wind Tunnel to investigate its noise reduction relative to a baseline fan of the same aerodynamic performance. The design objective of the QHSF was a 6 dB reduction in Effective Perceived Noise Level relative to the baseline fan at the takeoff condition. The design noise reduction was to be a result of lower levels of multiple pure tone noise due to the forward swept rotor, and lower rotor/stator interaction tone noise from a leaned stator. Although the design 6 dB reduction was observed in far-field measurements, the induct mode measurements revealed the reasons for this reduction were not the ones related to the design goals. All of the noise reduction was from the blade passing tone and its harmonics and most of this was unexpectedly from rotor/strut interaction modes. The reason for large differences in rotor/strut noise sources could not be determined with certainty. The reductions in the multiple pure tone noise for the forward swept rotor were not observed.

## INTRODUCTION

The NASA Advance Subsonic Technology program has an ongoing noise reduction element to provide the technology to meet increasingly restrictive airport noise regulations. As part of this effort a forward swept rotor fan was designed and constructed by Honeywell Engines and Systems for the purpose of reducing the noise of supersonic tip speed fans. This 22inch diameter fan, designated the Quiet High Speed Fan (QHSF), was tested in the NASA Glenn 9- by 15-foot Low Speed Wind Tunnel to investigate its noise reduction characteristics. In addition, a model of an existing conventional modern fan was also tested to provide a baseline for the purpose of comparison. The baseline fan incorporates many currently used noise reduction features. This paper will present measurements of both inlet and exhaust duct modes for the Blade Passing Frequency (BPF) and 2BPF tones in terms of PWL. In addition, the modal structure of the Multiple Pure Tones (MPT) in the inlet is presented.

## APPARATUS AND PROCEDURE

### Baseline Fan

The baseline fan is a 22-inch model of the fan used on the Honeywell TFE731-60 engine. This fan consists of a moderately aft swept rotor and an aft swept set of stator vanes. The baseline fan already has considerable acoustic design input. Blade/vane ratio, rotor-stator spacing and vane sweep were chosen with noise reduction in mind. A photograph of the baseline fan, with the fan case removed to better view the rotor and stator is shown in figure 1. Table 1 shows the stage design characteristics for both the baseline fan and QHSF. The design, takeoff, cutback, and approach tip speeds in feet/sec (and tip rotational Mach Number) are 1472 (1.32), 1328 (1.19), 1111 (0.99), and 868 (0.78) respectively.

### Quiet High Speed Fan

The QHSF was designed to have the same aerodynamic performance as that of the baseline fan (table 1) but with reduced noise. The acoustic design objective was a 6 dB reduction in perceived noise level relative to the baseline fan at takeoff condition. This noise reduction was to consist of reductions in MPT and rotor/stator interaction noise. The MPT noise is generally attributed to pressure disturbances from the shock structure on the rotor. The QHSF incorporates forward sweep on the rotor to reduce the relative velocity component normal to the blade leading edge to subsonic levels. The intent of this sweep is to eliminate the formation of the inlet shock and thus achieve MPT noise reduction. The goal of this rotor blade design was to contain any remaining shocks within the passage. This goal could not be met for all radial locations and rotor speeds but over a wide enough range so as to predict significant MPT reduction. A photograph of the QHSF with the fan case removed is shown in figure 1. The 50 degree forward sweep at the rotor tip is apparent.

One way to reduce rotor/stator interaction noise is to make the intersection of rotor blade wake the leading edge of the stator vane and the as close as possible to perpendicular. This lowers the trace speed of the wake intersection with the vanes in the circumferential direction, and causes multiple intersections per vane

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(phase cancellations). The QHSF employs significant lean to reduce rotor/stator interaction noise. Figure 2 compares photographs of the two vane sets to illustrate the difference in lean. The QHSF vanes have a 30 degrees lean at the tip while the baseline vanes have little lean. Reference 1 provides a more complete description of the QHSF design.

#### Aerodynamic Operation and Performance

The baseline and QHSF fans were operated so that they had the same bypass ratio speed line as the TFE 731-60 engine. A complete description of the model fan operation and performance can be found in reference 2. The goal was to reduce fan noise while matching, or improving, the performance capability of the baseline fan with respect to fan pressure ratio, mass flow, efficiency, and operability margin. The QHSF shows improved performance in most respects relative to the baseline fan, however, a part speed instability reduces operability margins to insufficient levels. The new fan stage had a design point peak adiabatic efficiency of 87.1% compared to 83.7% for the baseline fan. The operating line pressure rise at design point rotational speed was 1.770 and 1.755 for the QHSF and baseline fans, respectively. Weight flow at design point is 98.28 for the QHSF and 97.97 lbm/sec for the baseline fan. Unfortunately, the operability margins for the QHSF approached 0% at the part speed operating conditions near 75% speed. The baseline fan maintained sufficient margin throughout the operating range. Based on the stage performance measurements, this concept shows promise for improved performance over current technology if the operability limits can be solved.

#### Acoustic Mode Measurements

A continuously rotating microphone technique described in references 3 and 4 was used. The mode measurement system installed on the inlet is shown in figure 3. The same system installed in the fan exhaust is shown in figure 4. The rotating rake uses a control system slaved to the fan shaft to rotate at exactly 1/200th of the fan speed as if it were geared to the fan shaft. In the rotating frame of reference, each spinning circumferential mode order is Doppler shifted inversely proportional to its spin rate. Thus, each circumferential order is separated by 0.005 shaft orders in frequency. The radial order is determined by a least squares curve fit using the basis functions from the hard wall boundary condition of the Bessel's equation of all radial orders that might be expected, to the measured complex radial profile. In order to resolve the highest radial order that can propagate in the inlet, at 2BPF, 14 radial measurements were used, while only 8 were needed in the exhaust. These microphone signals are brought across the rotating frame by FM telemetry.

Several improvements in this mode measurement technique have been made since its first implementation reported in references 5 to 7. These improvements were

developed during tests on a large low speed fan rig (Active Noise Control Fan (ANCF)). The improvements involve the installation of aluminum foam windscreens over the microphones to lower self-noise, thus improving signal to noise ratio and additional foam shields on the exhaust rake (fig. 4) to attenuate the effects of the residual rotor wake interaction with the microphones. The locations of the mode measurement planes are also shown in figure 5. The inlet measurements were taken at the throat (minimum diameter). The exhaust measurements were taken in a plane just inside the nozzle exit.

#### Test Conditions

The fan models were run in the NASA Glenn 9- by 15-foot Low Speed Wind Tunnel at a Mach number of 0.10. The fan was operated at 11 different speeds for the mode measurements, as shown in Table 2 that include nominal, approach, cutback, and takeoff conditions. A slightly larger nozzle exit area was used for exhaust measurements to compensate for rake blockage. All tests were run with ducts in a hard wall (no acoustic treatment) configuration.

## RESULTS AND DISCUSSION

The complete modal structure (circumferential and radial orders) for BPF and 2BPF were measured. Both inlet and exhaust duct modes are presented in terms of sound power, PWL, referenced to  $10^{-12}$  watts. An example of the BPF modal structure for the exhaust is shown in figure 6 in the form of a 3-D bar graph. This figure is for the baseline fan at cutback power. The mode orders are displayed on the horizontal axes, and the power on the vertical axis. The back row represents the sum of the radial orders in each circumferential (m) order. This m-order power will be used to comparing the two fans for most of this report due to its simplicity. The rotor/strut interaction m orders are easily seen standing well above the extraneous modes (other than rotor/stator interaction modes). There are 10 struts in this fan, thus the interaction orders are 10 orders apart ( $m = 12, 2, -8, -18$ ). There are no rotor/stator interaction modes present for either fan at BPF due to their blade/vane ratio. Table 3 lists all the expected circumferential mode orders for both fans classified by source.

The mode results will be presented in the following order: the inlet 1 and 2 BPF, the exhaust 1 and 2 BPF and finally the inlet MPT.

#### Inlet Mode Power

A comparison of the QHSF to the baseline fan at the Blade Passing Frequency (BPF) tone is shown in figure 7. This plot shows the sum of all the strut interaction modes, the rotor locked mode ( $m = 22$ ), and total tone power as a function of fan speed. The rotating rake mode measurement used in this investigation can

suffer from interference due to the wake of the rake for the case of the inlet rotor locked mode ( $m = 22$ ). A discussion of this problem is presented in reference 8. When the  $m = 22$  mode is sufficiently stronger than this interference the measurement is accurate. In this case, PWL levels of approximately 125 dB are sufficiently high. There are no dramatic differences between the fans for this tone. The very rapid rise in tone power above 12,000 RPM is due to the rotor locked field,  $m = 22$  cutting on. Figure 8 shows this same comparison at 2BPF. In addition to the modes shown at BPF, the rotor/stator interaction  $m$  order ( $m = -8$ ) is cut on here, although it is generally below the 100 dB minimum of the plot. The trend in total tone power over the speed range is for the QHSF (forward swept rotor) to be a few dB lower. This trend is mostly due to the strut modes being lower for the QHSF, and at the higher speeds the  $m = 44$  rotor locked mode is generally lower also. It would not be expected that 1 and 2BPF tone power differences between fans would be as great in the inlet as in the exhaust since, the rotor transmission loss tends lower aft interaction sources relative to other sources. This is particularly true for the 2BPF rotor/stator interaction, which is a counter-rotating  $m$  order ( $m = -8$ ).

#### Exhaust Mode Power

A comparison of the modal power for the two fans at BPF is shown in figure 9. The QHSF has significantly lower total tone power than the baseline fan for most of the speed range. The primary reason for this is the enormous difference in the rotor/strut interaction modes. Differences of over 20 dB are seen in the strut mode power. The reason for these differences are not known but speculations as to the cause can be grouped into two categories; 1) weaker and or more leaned QHSF rotor wakes, 2) wakes weakened and or more modified by the QHSF stator. One wonders if the rotor/strut interaction levels for the baseline fan are unusually large and the difference between fans is thus exaggerated. Another mode where huge differences are seen is the  $m = 22$  rotor/stator interaction. Here the difference between fans approaches 25 dB. Possible explanations for this could be weaker QHSF rotor wakes (longer rotor/stator spacing at the tip) or perhaps more likely the greater lean of the QHSF stator vane at the tip (fig. 2).

The 2BPF comparisons are shown in figure 10. The total tone power for the QHSF is generally 5 dB lower than the baseline fan over the full speed range. This is primarily a result of the reduced levels of the rotor/stator interaction mode,  $m = -8$  in the QHSF. These levels are 7 to 14 dB lower for QHSF. Differences in the strut modes in favor of QHSF of 1 to 10 dB also contribute a small amount to the overall difference. Just as with the  $m = 22$  at BPF the  $m = 44$  at 2BPF rotor/stator interaction is generally much lower for the QHSF. Just as with the BPF results, the 2BPF

results the longer tip spacing of the swept rotor, and the extreme lean of the stator at the tip are likely causes of this benefit. In recent tests (as yet unpublished) the baseline rotor was run with the QHSF stator. Far-field results indicated tone noise levels similar to those of the QHSF. This indicates the leaned stator is probably responsible for much of the rotor/strut and rotor /stator source noise reductions in the QHSF.

#### Multiple Pure Tones

The MPT power distribution in the inlet, at five different speeds is shown in figure 11. Shaft orders from 1 to 22 are presented, although the order 22 is the BPF tone, it is included for reference. Both fans have similar power distributions, considering that the individual tones are related to blade to blade manufacturing differences. There appears to be a trend in the power distributions, for the maximum power to occur near the low end of the propagating shaft orders (near cutoff).

The sum of the power for all the shaft orders of the MPT was called MPT power and is shown in figure 12(a) for the supersonic rotational tip speed range. The baseline fan has a slight advantage over the QHSF but not over the full range of speed. This MPT power levels for the QHSF seems disappointing in light of the design goals. It was always recognized that the blade shock could not be swallowed for the full speed range of fan speed but, at least, there might be an advantage at the low supersonic tip speeds. This was not the case as shown in figure 12(a), but the swept rotor (QHSF) was slightly higher in MPT power at the low supersonic speeds. Figure 12(b) shows the comparison of the two fans when the MPT power is added to the BPF power. The motivation for this plot is the similar nature of these sources, rotor locked modes. The nearly linear relation between rotor speed and power is interesting. Since the inlet BPF tone (mode) is a result of the 22 blade direct aerodynamic field of the rotor, perhaps the MPT sources should be viewed in the similar way as the FFT of the complete field.

### **CONCLUDING REMARKS**

Although the design noise reduction goal for the QHSF of 6 dB in EPNL was met at takeoff (ref. 9), unexpectedly the reduction seems to come mostly from the 1 and 2BPF tones and not the MPT. The mode measurements show the vast majority of these reductions come from rotor/strut interaction. The QHSF was expected to reduce rotor/stator interaction noise and it did but, both the reductions of this source and the rotor/strut source seems unusually large (20 to 24 dB in places). One wonders, if these very large reductions might, in part, be a result of the baseline fan being somewhat noisier than expected, especially in regard to the rotor/strut interaction source. Recent tests of the combination of the baseline rotor and QSHF (leaned)

stator indicate the stator may be responsible for the large rotor/strut noise reduction. While the reductions in 1 and 2 BPF tone power level are impressive, the expected significant reductions in MPT levels for the QHSF in this investigation were not observed.

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Table 1. Fan design parameters at fan aero design point for 22-inch diameter fan (baseline and QHSF)

Parameter	Value
Corrected weight flow, lbm/sec	98.9
Corrected weight flow per unit area, lbm/sec/ft <sup>2</sup>	42.7
Tip speed, ft/sec	1474
Bypass ratio	3.83
Overall pressure ratio	1.82
Adiabatic efficiency, overall, %	89.5
Fan hub/tip ratio	0.35
Rotor blade count	22
Stator vane count	52

Table 2. Fan test points

Fan corrected rpm	Fan corrected tip speed, ft/sec	Fan tip rotational Mach No.
8516	817	0.731
9039 (approach)	868	0.777
9510	913	0.817
10646	1022	0.915
11150	1070	0.958
11572 (cutback)	1111	0.995
12500	1200	1.074
13342	1281	1.147
13831 (takeoff)	1328	1.189
14500	1392	1.246
15000	1440	1.289

Table 3. Circumferential mode orders for fans cut-on at the highest speed

Harmonic	Rotor (22)/stator(52)	Rotor(22)/strut(10)
1 BPF	22	-32, -28, -18, -8, 2, 12, 22
2 BPF	-8, 44	-56, -46, -36, -26, -16, -6, 4, 14, 24, 34, 44, 54, 64



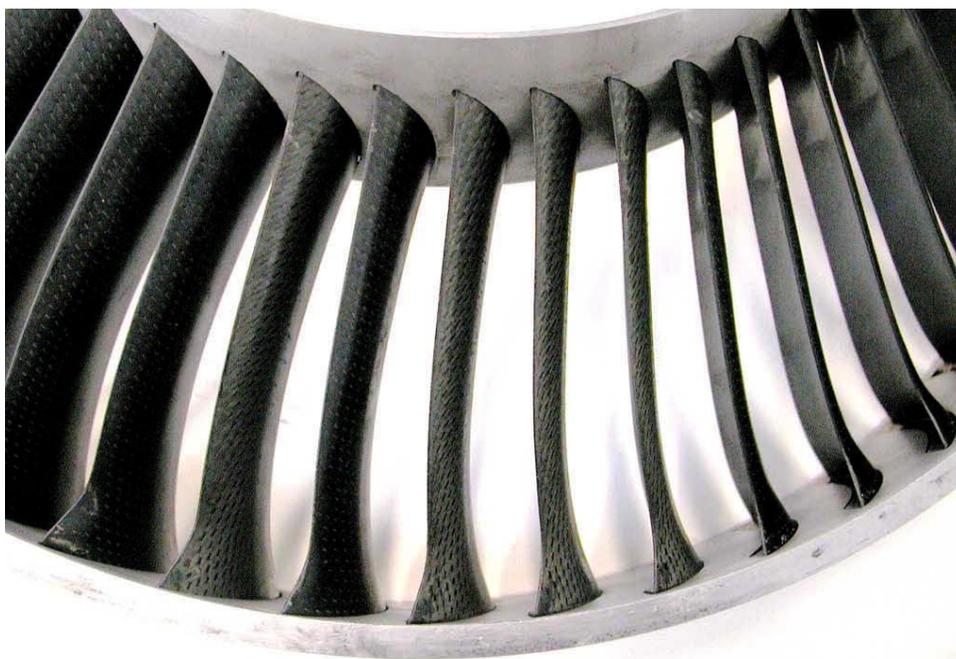
Baseline



Quiet High-Speed

Figure 1. Photographs of the model fans with their casings removed.

Baseline



QHSF



Figure 2. Photographs of the stators of the model fans.

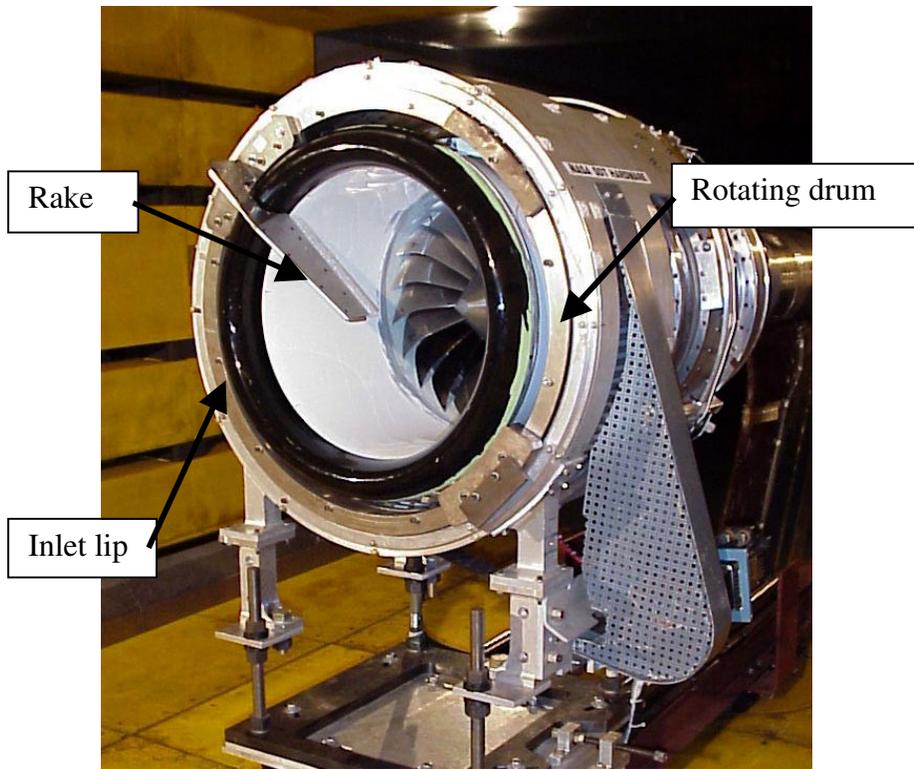


Figure 3. Mode measurement system installed on the inlet of the QHSF.

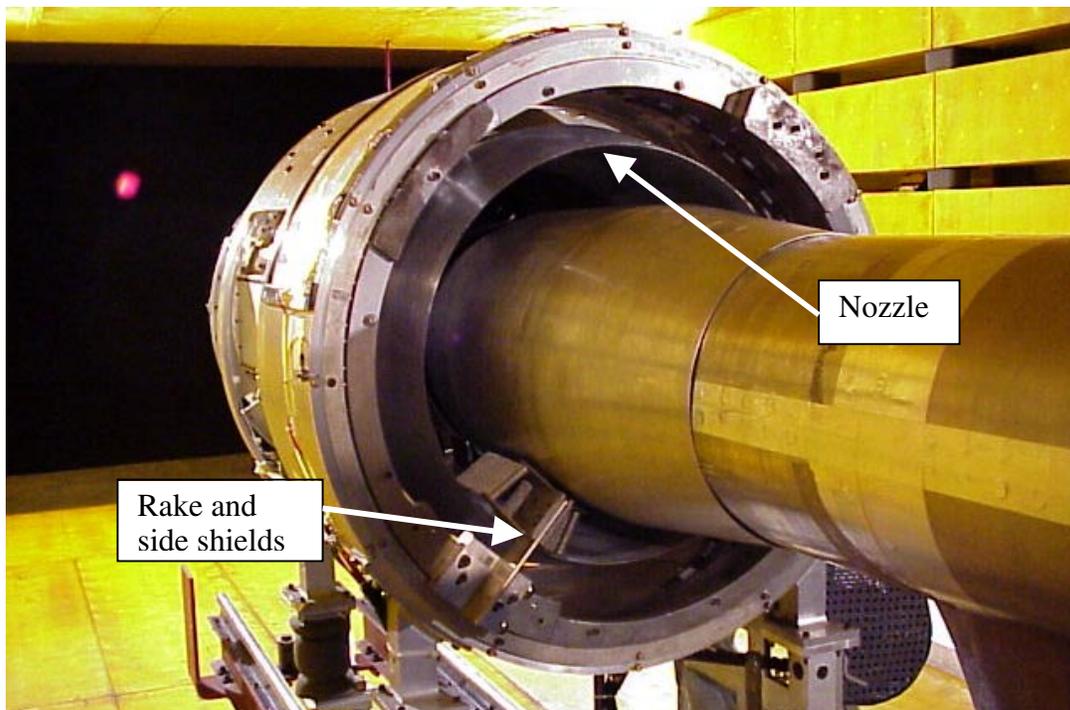


Figure 4. Mode measurement system installed of the exhaust.

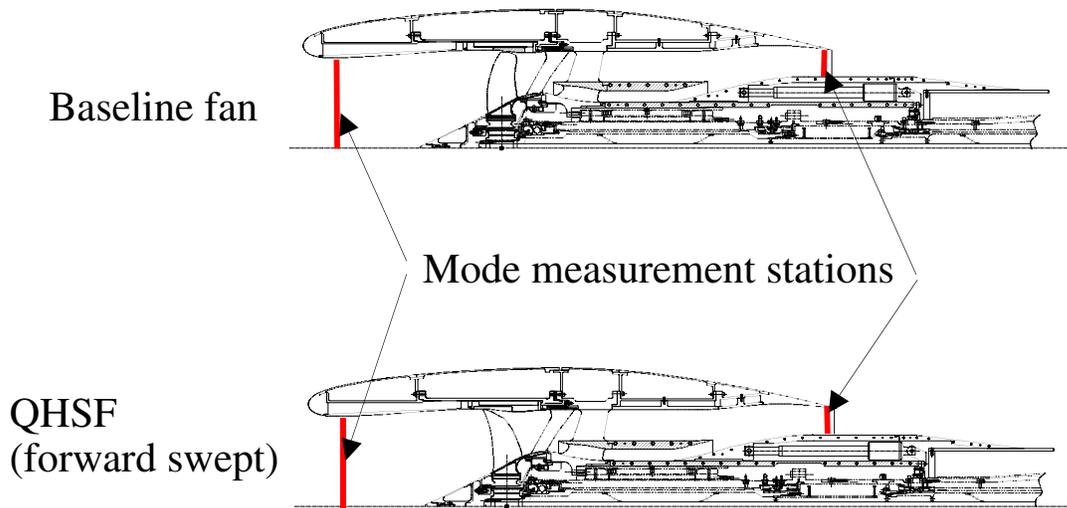


Figure 5. Fan model cross sections showing mode measurement stations.

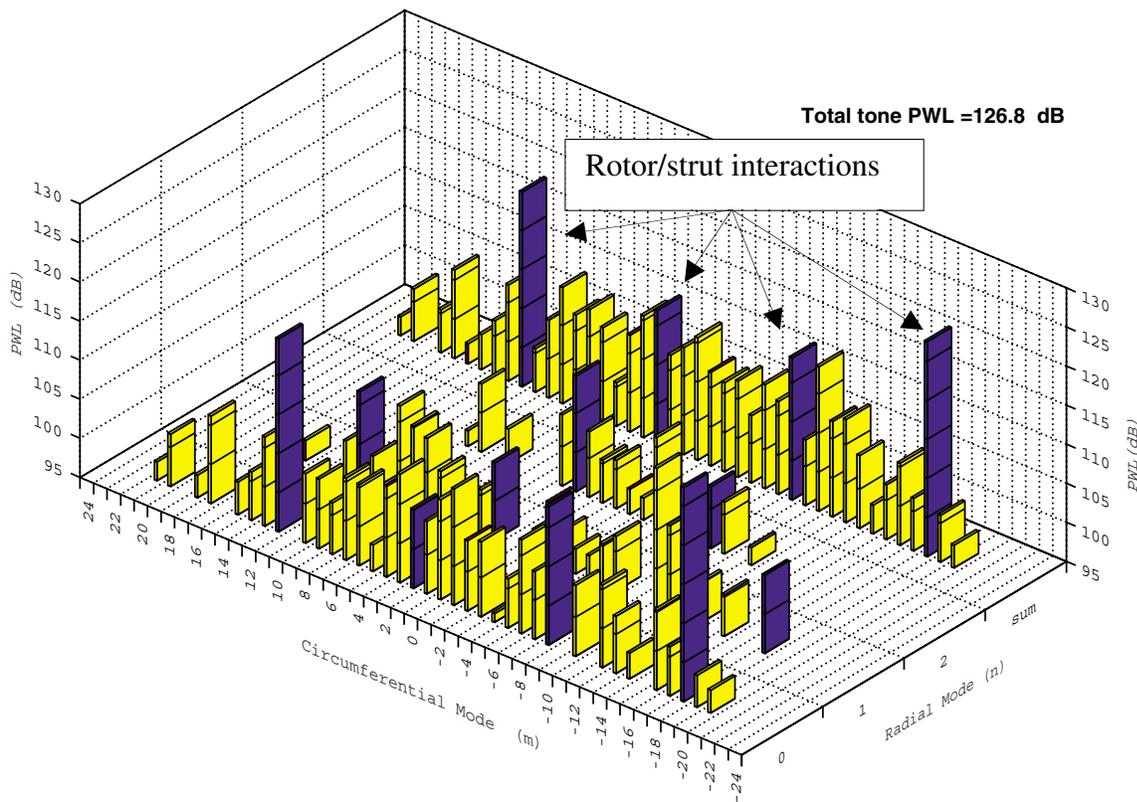


Figure 6. Modal structure for the baseline fan in the exhaust at BPF, cutback power (11570 RPMc).

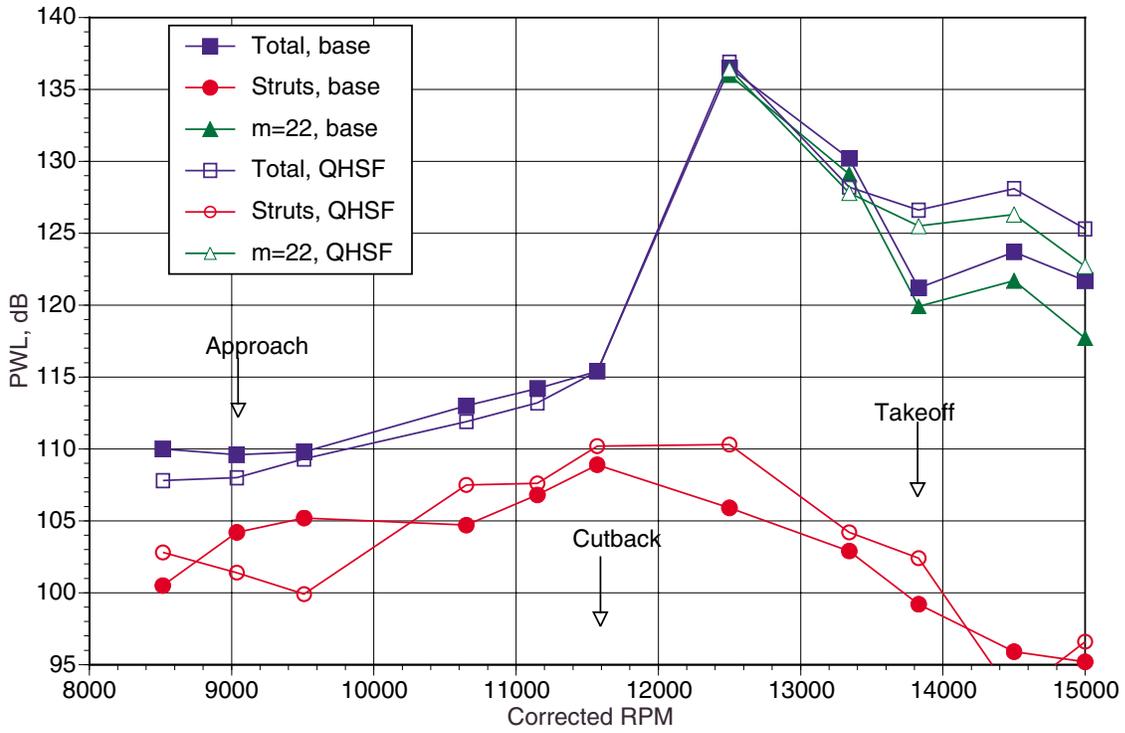


Figure 7. Comparison of the inlet modal power of the baseline fan to the QHSF at BPF.

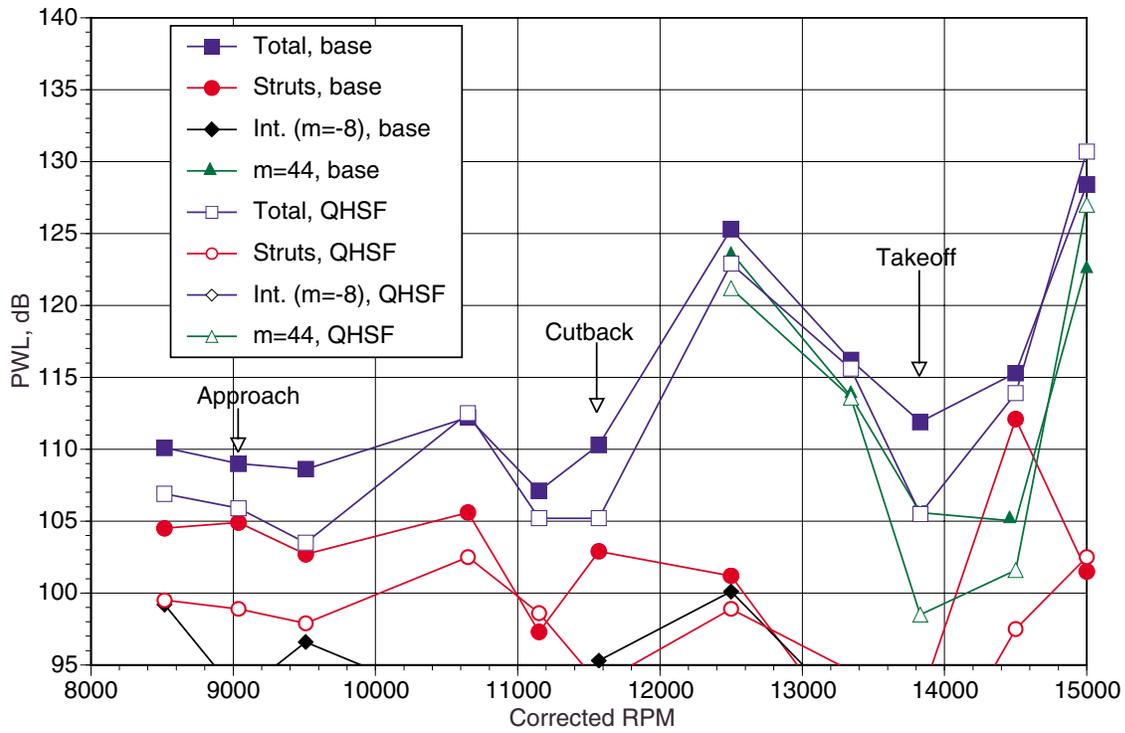


Figure 8. Comparison of the inlet modal power of the baseline fan to the QHSF at 2BPF.

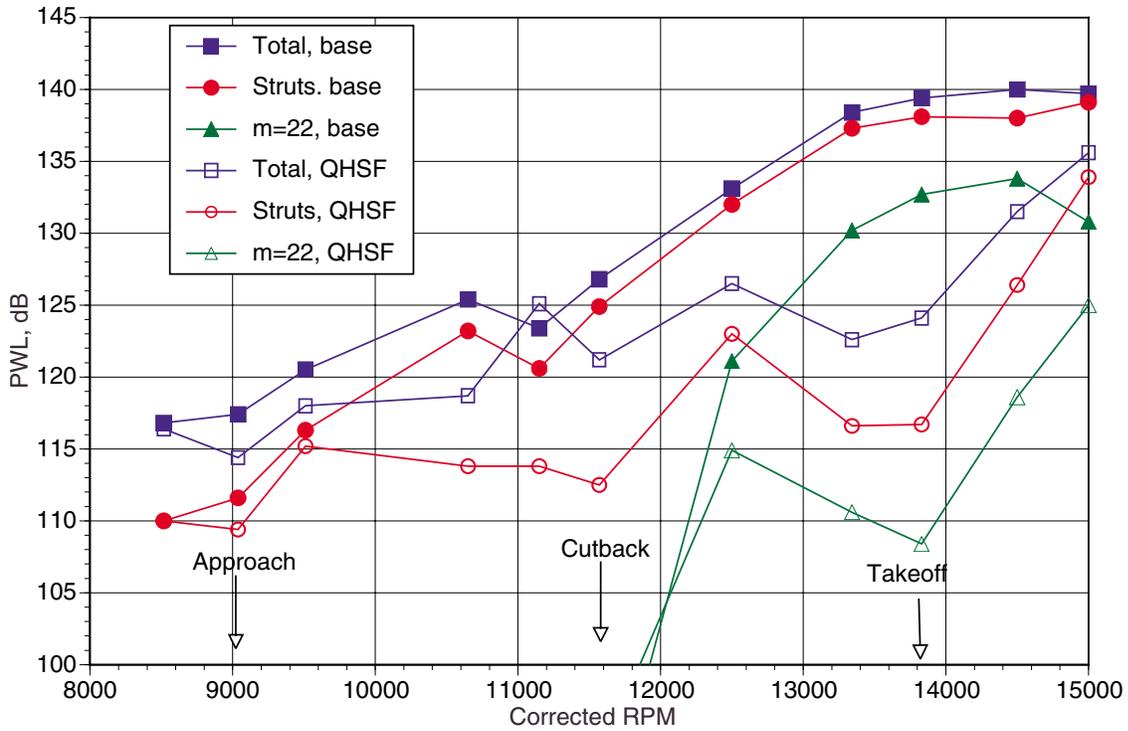


Figure 9. Comparison of the exhaust modal power of the baseline to the QHSF at BPF.

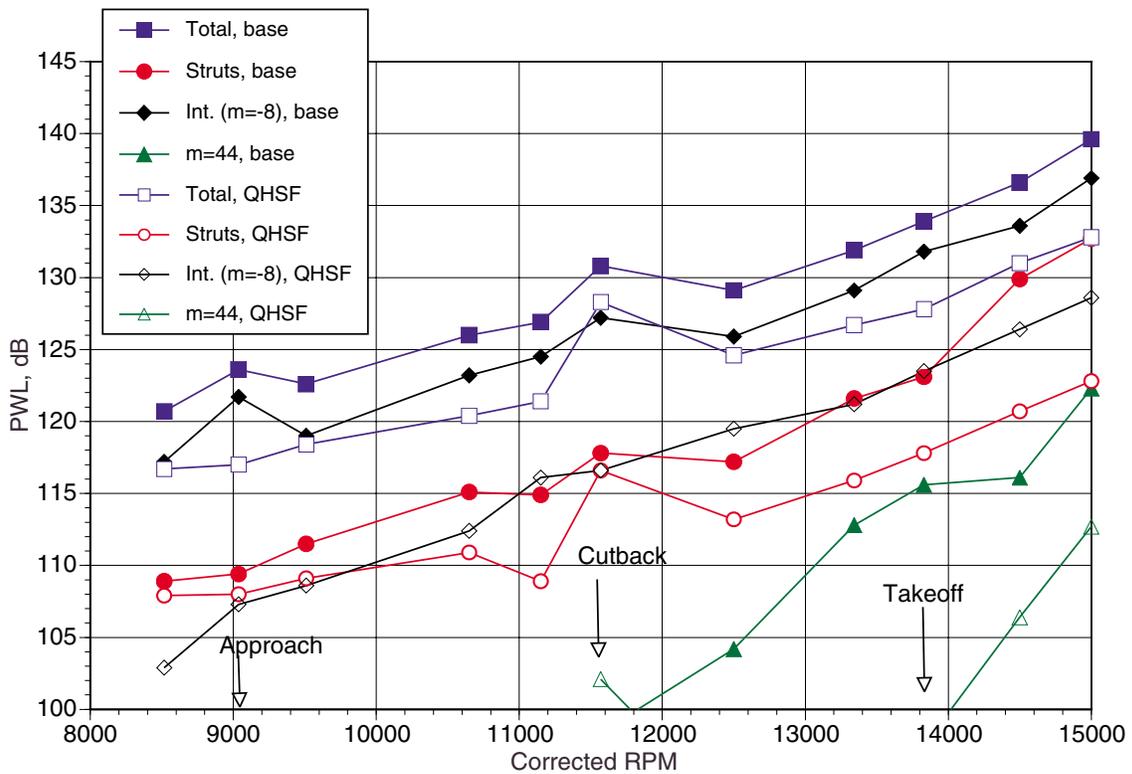


Figure 10. Comparison of the exhaust modal power of the baseline fan to the QHSF at 2BPF.

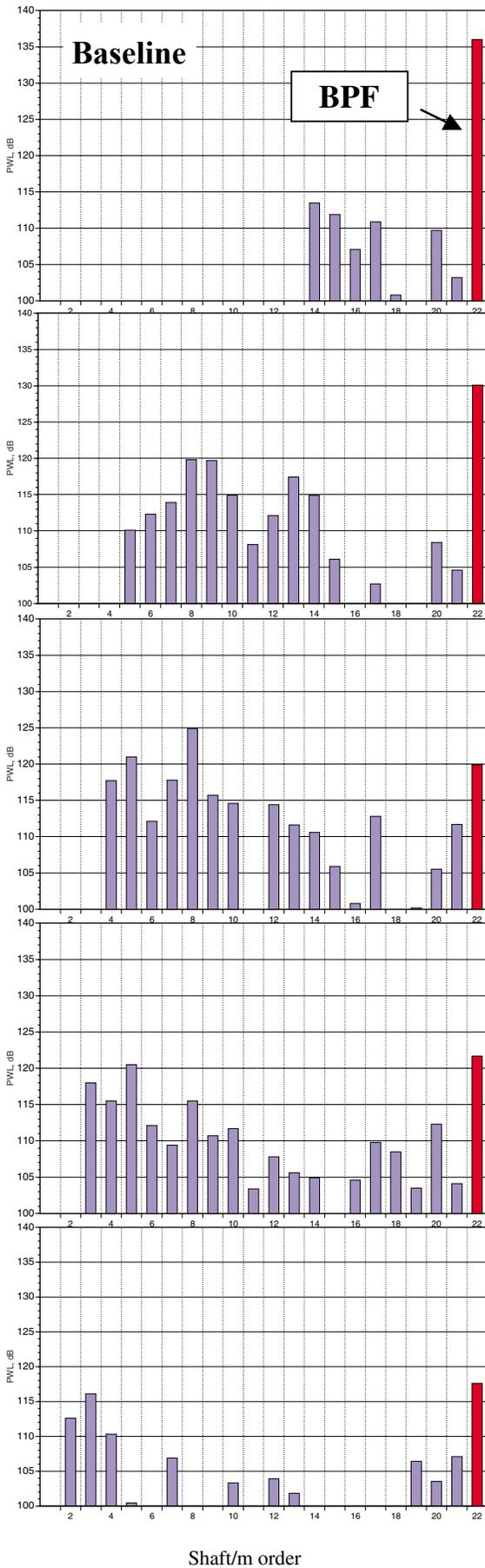
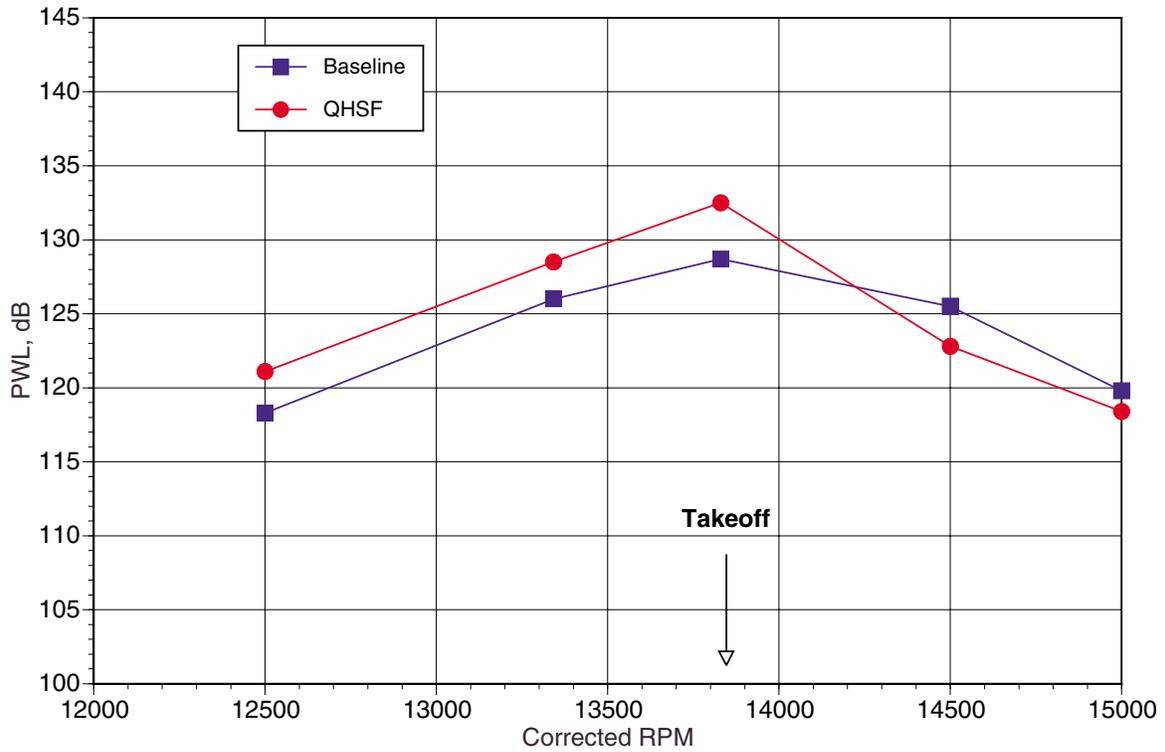
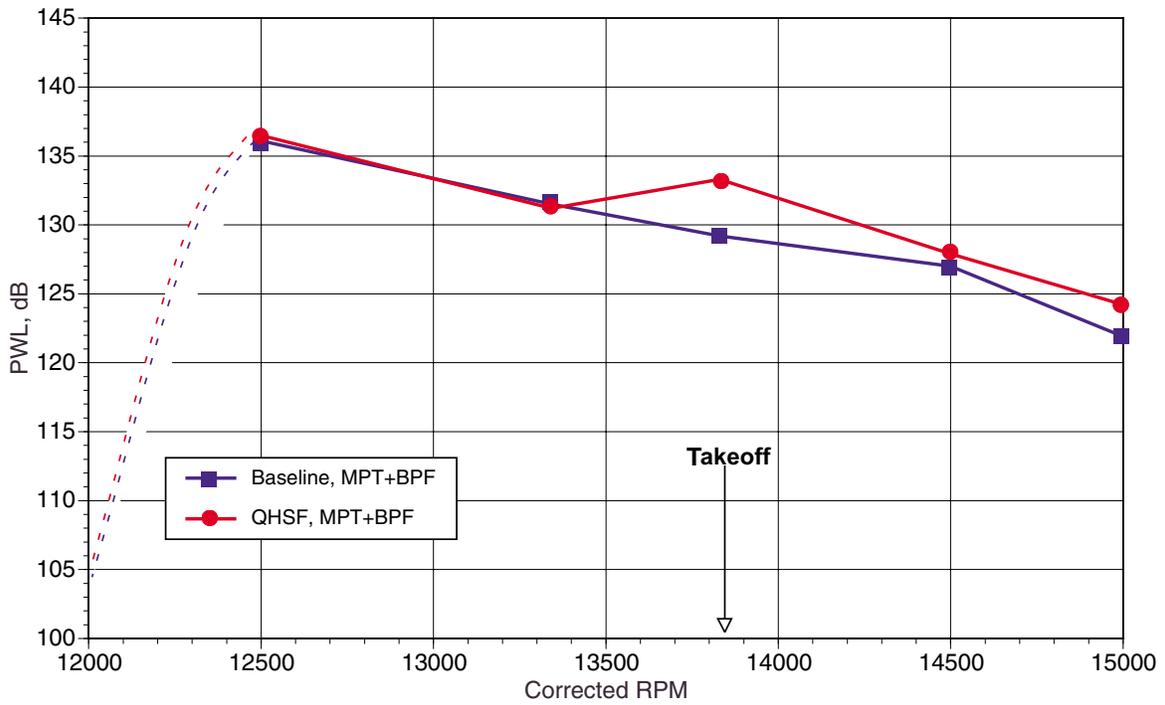


Figure 11. Comparison of MPT shaft order power distribution between the baseline and QHSF.



a) MPT power.



b) MPT + BPF power.

Figure 12. Comparison of MPT power (sum of all cuton modes up to  $m=21$ ) and MPT plus BPF power for the baseline and QHSF.



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